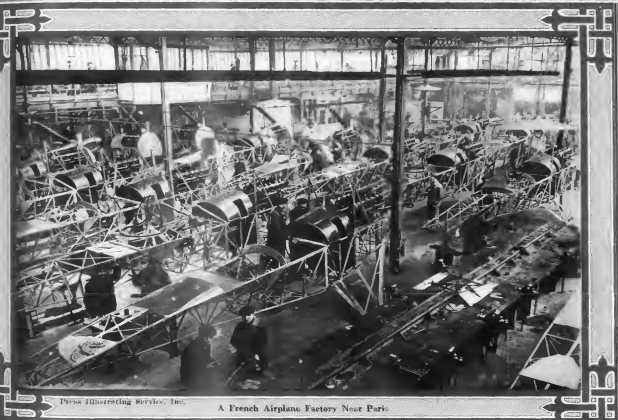


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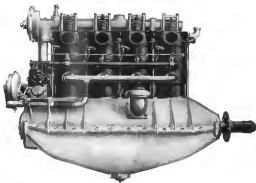


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## AVIATION AND AERONAUTICAL ENGINEERING

VOL. II. NO. 6

APRIL 15, 1917

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HERBERT M. WILLIAMS, B.S.

Vol. II

April 15, 1917

No. 5

**I**T is a generally admitted fact that the Navy Department is not only placed on the present application for training from civilians there who cannot fly and the War Department is in a similar predicament. Civilian schools are assisting in the training of young men who are desirous of entering the service, but the facilities of these schools are entirely inadequate for the training of the 4000 civilians which the National Advisory Committee for Aeronautics has estimated as necessary for our Army and Navy.

The War Department already has more applicants on its lists than it is capable of accepting, and it is not possible to train the surplus. Literally thousands of applicants for examination for enlistment in the United States Reserve Corps are rejected by the War Department. They find the output of mechanics is enormously increased, these applicants cannot be asked upon. It has recently been publicly announced that all the airplanes and airplanes which the Army and Navy together can expect to receive before January 1, 1918, are limited to about 500.

Another limitation upon the Government's ability to train these lies in its inability to get a large corps of competent instructors. About two capable per instructor has been found in experience to give the best results. But the Government's corps of flying instructors is unfortunately small. It has been found that more than one-half the civilians there who have sought positions as civilian instructors for the Government were unable to give satisfactory instruction in aviation.

The rank of young men anxious to learn to fly, has not been equalled by any rush of mechanics anxious to try for airplanes. Civilian aircrafts to learn to fly are out of the thousands into the hundreds. Mechanics trained in the use of aeronautical equipment are literally scarce. Both the Army and Navy in the regular establishment as well as in the Reserves are anxious to secure first class mechanics, carpenters and engine men. The mechanics of an air squadron are almost as responsible for the squadron's success or failure as the aviators. With literally thousands of applicants on its lists from civilians who ask the Government to teach them to fly, there is a noticeable shortage of mechanics of the best type in the Army, the Navy, the Naval Reserve and the Signal Enlisted Reserve Corps.

But even more important and discouraging than the shortage of flying instructors and of mechanics is the limited output of mechanics. The output of airplanes must be speeded up. In the first place, standardization of parts must be given immediate consideration. It has been on the tip of the tongue wherever aeronautics was

discussed for the past year. To date, however, practically no tables of standard parts are available and even where tables are available engineers and draftsmen are reluctant to utilize standard equipment.

The second thing that can be done to speed up production is the placing of a cash deposit of 25 per cent of the purchase price with every order which the Government places for aeronautical material. At the present time this is specifically forbidden by the laws of the United States. However, it is a practice employed at all daily business by every Government of which it is not except the United States. It is of growing importance that the practice be instituted in this country especially for increasing the output of aeronautical material. Only Congressional legislation will make this possible.

AVIATION AND AERONAUTICAL ENGINEERING, does not predict that the adoption of standards by the manufacturers or the adoption by the Government of the practice of making part payments in advance will result in getting a sufficient supply of airplanes for the United States. But if the two steps outlined are taken it will be a definite help.

Naturally, mechanics desiring information about entering the Regular Service or the Signal Enlisted Reserve Corps for aviation duty should apply to the Chief Signal Officer, United States Army, Washington, D. C. If they wish aviation duty with the Navy either in the regular or reserve forces they should apply to the commandant of the Naval District in which they reside.

### U. S. A. Aeronautics

In this issue, our readers will find a report from the Massachusetts Institute of Technology on the recent tests conducted there on a number of wing sections developed by the Aviation Section of the Signal Corps, U. S. A. The announcement of these sections marks a distinct advance in aeronautical practice. In using the Eiffel and H. A. F. sections which have been almost universally employed hitherto, constructors have had to pay the penalty for efficiency and good lifting power by more or less successfully conducting structural difficulties.

The development of these sections will go a long way to remove the incompatibility hitherto present between aerodynamical excellence and practical facility of construction. It is too early to say whether some of these sections are altogether superior to any well known wing-curves as those developed by Eiffel and the H. A. F., but further comparative tests are in progress and the indication is that the new wing sections will mark a decided improvement.

## Six United States Army Wing Sections

By Captains Edgar S. Garrell and H. S. Martin, U. S. A.\*

These wing sections developed by the Aviation Section of the Signal Corps, after considerable study, have been an aerodynamic and a structural point of view. Developed partly from the latter point, they have proved to be efficient and show very satisfactory lift coefficients.

## Structural Development of the Section

In Fig. 1 are shown the dimensional outlines of the six sections tested. Some of the considerations involved in developing the sections are interesting.

The U. S. A. 1 is a modification of the Clark wing foil (relationships and dimensions of which are described in Hunsaker's "Elements of Stability of Aeroplanes"). This was an excellent high speed wing with a maximum lift drift of 18. By using the depth at the position of rear spar, it was made structurally much more practical, the maximum lift was increased and the maximum lift drift only reduced very slightly.

The U. S. A. 2 has the same upper surface as the R. A. F. 3 but the lower surface has been modified and deepened from a structural point of view, without any loss from the aerodynamic point of view.

The U. S. A. 3 and U. S. A. 4 are both modifications of the U. S. A. 2. In the first section, the nose of the 30-in chord has been moved forward 34-in and the inclination of the first fifth of the 30-in chord spread out accordingly. In the U. S. A. 4, the nose of a 30-in chord has been moved back 34-in and the inclination of the first fifth of the 30-in chord rounded out accordingly.

The U. S. A. 5 was carefully developed from both structural and aerodynamic considerations, with very satisfactory results.

## Results of Tests

The tests were conducted under the standard conditions. Results for  $K_y$ ,  $A$ ,  $A'$ ,  $L$ ,  $D$  and center of pressure position are given in Table I, and in the curves of Figs. 2, 3 and 4. The wind tunnel speed was 30 m. p. h., in every case, and the models 16-in. span by 3-in. chord; the dimensions generally employed, and useful so that some for purposes of comparison.

The National Physical Laboratory results, as published in the British Reports, are based on L.F. (chord of wing in feet  $\times$  velocity of relative wind in feet second) values of 6.3. In Edgley's laboratory, with greater wind speeds the values of L.F. range from 16 to 40. The Institute tests are conducted with an intermediate value of 11 for L.F. It is, therefore, not possible to make strictly accurate comparisons without further tests which are now in progress at the Institute. It is, however, perfectly clear that, apart from the good aerodynamic features, the sections have remarkably good structural properties.

In Table 2 comparative figures for the six sections are set out in accordance with the plan employed, as the Course in Aerodynamics and Airplane Design (AVS 200, Aero-Armory, U. S. Army, October 1, 1936), together with values for the R. A. F. 3, U. S. A. 2, 3, 4, and Refl 32; the drift and  $L/D$  values for the latter being subject to revision as stated on completion of above tests.

For ease of the wings close the maximum  $K_y$  fall below 0.05, except at high angles, as for U. S. A. 6. For all the wings, there was a range of working angles. None of them exhibit a sharp drop at the stall point. With the exception of U. S. A. 4, they all

have good maximum values of  $L/D$  at the stall. The  $K_y$  or  $L/D$  values at the onset of gross stall, as in Fig. 4, it is seen that some of the wings may be said to be shockingly stable, but none of them lose a sudden reserve of pressure ratios within the usual stall range.

TABLE 2

Airfoil	Lift coefficient at wing tip to 10 in. span at 10 m. p. h.	Maximum $K_y$ at 10 in. span			Maximum $A$ , $A'$ , $L$ , $D$ at 10 in. span		
		Angle of incidence	$K_y$	$L/D$	Angle of incidence	$A$	$A'$ , $L$ , $D$
U. S. A. 1	10	1.0	0.0412 (17.5)	1.2	0.0116 (9.8)	0.0116 (9.8)	0.0116 (9.8)
U. S. A. 2	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)
U. S. A. 3	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)
U. S. A. 4	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)
U. S. A. 5	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)
U. S. A. 6	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)

TABLE 3

Airfoil	Lift coefficient at wing tip to 10 in. span at 10 m. p. h.	Maximum $K_y$ at 10 in. span			Maximum $A$ , $A'$ , $L$ , $D$ at 10 in. span		
		Angle of incidence	$K_y$	$L/D$	Angle of incidence	$A$	$A'$ , $L$ , $D$
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U. S. A. 6	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)	0.0117 (9.8)

with modern air-lifting test surfaces, no difficulties would be met in carrying out such tests.

It is worth noting that the U. S. A. 4 has a higher maximum lift coefficient than almost any wing section tested, — 0.0416 compared with 0.0411 of the R. A. F. 3. It should have an  $L/D$  of approximately the same value and so far better structural. This wing would seem to be particularly suitable for very heavy machines such as bombers.

The U. S. A. 1 has a higher lift than Refl 32—about 0.0412 compared with approximately 0.0404 and its maximum  $L/D$  of 1.8 would in the Refl 32 probably exceed the 1.52 of the P-51 wing. Although for small values of  $K_y$  and high speeds, the Refl 32 has better efficiency, U. S. A. 6 also with a higher maximum lift, would require most favorably in a high speed machine with very high wing at which data has little to be published. The present results show the change in the U. S. A. 3 and 4 would be very close. The U. S. A. 1 would give a better lift speed and  $L/D$  at high angles of incidence at low angles and high speeds.

The U. S. A. 2 and 3 would be good all round wings, apparently slightly better than the R. A. F. 6.

An interesting feature of the series is the successful employment of lower wings on upper and lower surfaces. It has been generally accepted that large number meant increase in lift with corresponding decrease in efficiency, while highly rounded wings would give high efficiency but low lift. With shiffling design, it is apparently possible to retain both good features.

## AERODYNAMIC LABORATORY TESTS

TABLE 1

Lift coefficient at wing tip to 10 in. span at 10 m. p. h.	Maximum $K_y$ at 10 in. span			Maximum $A$ , $A'$ , $L$ , $D$ at 10 in. span		
	Angle of incidence	$K_y$	$L/D$	Angle of incidence	$A$	$A'$ , $L$ , $D$
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U. S. A. 5	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)
U. S. A. 6	10	1.0	0.0416 (18.0)	1.3	0.0117 (9.8)	0.0117 (9.8)

Lift coefficient at wing tip to 10 in. span at 10 m. p. h.	Maximum $K_y$ at 10 in. span			Maximum $A$ , $A'$ , $L$ , $D$ at 10 in. span		
	Angle of incidence	$K_y$	$L/D$	Angle of incidence	$A$	$A'$ , $L$ , $D$
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\* By permission of Professor W. H. Phillips, Professor of Naval Architecture, in Charge of Department of Naval Architectural Engineering, University of Cambridge, England, with a N. S. S. and T. H. S. S.

U. S. A. 1 is based on 10 in. span, U. S. A. 2 is based on 10 in. span, U. S. A. 3 is based on 10 in. span, U. S. A. 4 is based on 10 in. span, U. S. A. 5 is based on 10 in. span, U. S. A. 6 is based on 10 in. span.

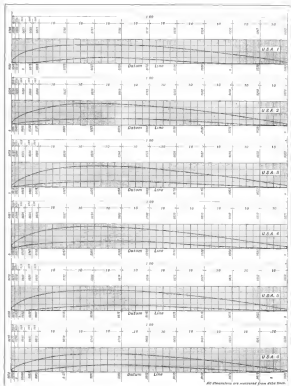


FIG. 1

All dimensions are measured from 45th inch.

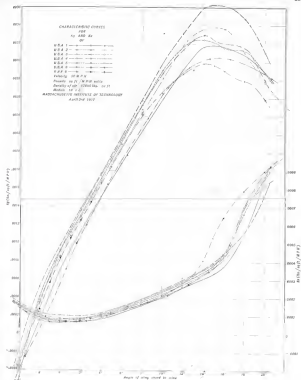


FIG. 2





highly channelled out fed by longitudinal cleave. The tensile fiber strength of wood is much in excess of the compressive stress, but even if the compressive stress of wood is employed in the formula  $f = \frac{M}{I}$ , it is no true estimate. If the

formula is employed for strength calculations on bending, it is assumed that the material is still behaving elastically up to actual failure, and therefore that the fiber stress is still directly proportional to the distance of the fiber from the neutral axis. As a matter of fact, the elastic limit of the material may have long since passed when the elastic limit is reached, the wood may have shifted, and the stress may be no longer proportional to the bending. Therefore, in stress calculations for wood, a safety factor should always be applied, making use of the modulus of rupture value which values are given in Table 2—deduced from actual bending tests, which are for more trustworthy grades.

#### Knots

Knots—Knots in the timber cut from the stem or branches of a tree because of the encroachment of a limb, either broken or shed, by the subsequent annual layers of wood. Most knots originate at the path of the stem, and the knots formed there in a log are therefore small, increasing in size toward the butt, the log as the tree is growing, the layers of wood are a continuation of those of the stem. But a majority of the knots die after a time, and if a portion of a dead limb is left, partially removed by the growing stem, there will be no intimate connection between the new stem wood and the dead wood at the knot, and it is bound or cut in the path of the growth of the log will contain a loose knot. A knot cut from the log at such a depth that the knot is intersected at a point where it was removed while still living will contain a sound knot, unless the knot has rotted, become badly checked, or contains a large pitch cavity.

A sound knot is usually harder than the surrounding wood, and in continuous loads it is to be very common. On the other hand, it may constitute a defect because of its non-resistance to gnaw or warping. Otherwise it constitutes a defect only on account of the disturbance to the grain and difficulty caused in working, or in the event of its occurrence on the outer side of a timber used as a beam, a weak joint exists, owing to its small resistance to tensile stress. A knot constitutes an impediment to the splitting of timber, since the fibers of the stem wood above a knot bend and curve around the knot, while the fibers below may continuously enter the knot. This often happens that a split started above a knot will never run into a knot, but one started below is very apt to do so.

#### The Effect of Moisture on Strength of Wood

Loss of moisture does not affect the strength of wood in any way, but the total moisture content has been reduced below the critical grain shrinkage, which represents the fiber-saturation point. Beyond this point, a constant loss of moisture affects the strength very considerably. Thus the strength of green wood is only 80 to 85 per cent of normal air-dry conditions (12 per cent moisture), while the strength of kiln-dry wood exceeds the strength of air-dry wood by some 20 to 70 per cent.

#### Time Factor in Tests of Timber

Timber differs from most other materials in that small variations in the rate of application of load have a more pronounced effect upon the strength and stiffness shown by a specimen under test. If a timber specimen, when a load is loaded rapidly, it will appear to have a higher elastic limit and ultimate strength, and will also appear to be stiffer, than if it will be loaded less rapidly. This is due to the fact that the deformation lags far behind the load, and if very low load is permitted to remain upon a specimen for a time the deformation increases, the amount of increase becoming greater for longer periods of time. Actual failure appears to be consequent upon the attainment of a certain limiting amount of deformation or stress, rather than a limiting load or strain.

#### Difficulties of Wood Construction in Airplanes

The comparative values of Table 2 demand the most careful study. A certain type of timber may be most suitable for the direct stress to which it is subjected, yet fail completely

under certain indirect stresses, or for indirect, at the construction or due to faulty design. For example, at the loading of a wing spar to the bolts, if the bolts are not correctly placed, they may shear out the wood. These points will be covered in detail in this design, and the engineer will have to know the value of working not only the direct stresses in point of timber is a machine, but also the indirect stresses producing twisting across the grain, shear, etc.

#### Strength Values for Timber

In no material are such conflicting values given by various authorities as for timber. The use of the specimens under test, the direction, the method of applying the load, and the previous history, all tend to introduce discrepancies. Until the Bureau of Standards, or some other testing laboratory, has gone the length into the question, all the values employed by various researchers, if it is open to comparison. Table 2 is a collection of information taken from available sources. This table is not unacceptably high if representative clearly values, and a correct guide. In any case, design, other stress is not taken as a constant, without due consideration of the modulus of rupture.

Species.	Tensile strength, psi.		Compression, psi.		Modulus of rupture, psi.		Modulus of elasticity, psi.		Shrinkage, percent.		Weight, lb. per cu. ft.	
	Parallel to grain.	Perpendicular to grain.	Parallel to grain.	Perpendicular to grain.	Parallel to grain.	Perpendicular to grain.	Parallel to grain.	Perpendicular to grain.	Parallel to grain.	Perpendicular to grain.	Parallel to grain.	Perpendicular to grain.
Aspen	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Balsam	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Birch	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Cedar	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Fir	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Hardwood	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Maple	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Pine	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Redwood	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
Spruce	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35
White Pine	11,000	1,000	5,000	1,000	1,400	400	1,500,000	1,500,000	10	10	35	35

Adapted from report of the Forest W. H. Keith for collection from the forest notes on timber.

#### Wires and Cables

The following terms are in common use: (1) "Solid steel wire" or "solid wire" at one wire of uniform diameter. (2) "Stranded wire" consisting of 7 or 19 wires twisted together and known to the trade as "strand twisted." (3) "Steel" or "rope wire" consisting of 7 strands twisted together, forming a rope, the strands being called 7 wires or 7 strands. (4) "Flexible wire" composed of steel strands of same size, with a center of either wire or wire, as indicated. The steel with the center wire, is considerably more pliable than that with the wire center.

Vanadium steel and other special steels have not as yet become established as desirable wire steels, and carefully not high grade carbon steel is at present most largely employed in the manufacture of wires and cables.

#### Properties of Metals

Only the broadest outlines can be given of the metals that are commonly employed in airplane construction. To enter into any adequate description of this branch of the work would require a book in itself. The construction must keep constantly before him some standard book on the subject and at the same time must be able to select the material and parts which are most suitable for the work to be employed in his design.

The following table of weights and melting points for various metals may be of service:

Material	Weight per cubic inch	Weight per cubic foot	Specific gravity	Melting point, degrees Fahrenheit
Aluminum	2.7	168	2.7	1,200
Copper	8.5	528	8.5	1,900
Iron	4.9	302	4.9	2,500
Steel	4.9	302	4.9	2,500
Brass	5.2	325	5.2	1,600
Lead	7.3	458	7.3	1,200
Mercury	0.49	30	13.6	357
Gold	19.3	1,200	19.3	2,400
Silver	10.5	655	10.5	1,700
Platinum	21.5	1,340	21.5	3,000
Antimony	4.4	275	4.4	1,100
Phosphorus	0.0022	0.14	0.0022	220
Carbon	0.0004	0.025	0.0004	3,600
Graphite	0.0004	0.025	0.0004	3,600
Asbestos	0.0004	0.025	0.0004	3,600
Flint	0.0004	0.025	0.0004	3,600
Quartz	0.0004	0.025	0.0004	3,600
Rock salt	0.0004	0.025	0.0004	3,600
Ice	0.0004	0.025	0.0004	3,600
Water	0.0004	0.025	0.0004	3,600

TABLE 2

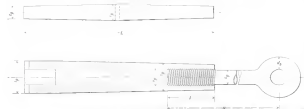
#### Strength and Weights for Wire and Cable

STEEL WIRE		
Reinforcing Steel Wire		
Diameter of wire (inches)	Breaking strength (lb. tensile)	Approximate weight per 100 ft., lb.
1/16	100	0.1
1/8	400	0.4
3/16	900	0.9
1/4	1,600	1.6
5/16	2,500	2.5
3/8	3,600	3.6
7/16	4,900	4.9
1/2	6,300	6.3
5/8	7,900	7.9
3/4	9,600	9.6
7/8	11,400	11.4
1	13,200	13.2
1 1/8	15,100	15.1
1 1/4	17,100	17.1
1 1/2	19,100	19.1
1 3/4	21,200	21.2
2	23,300	23.3
2 1/4	25,400	25.4
2 1/2	27,500	27.5
2 3/4	29,600	29.6
3	31,700	31.7
3 1/4	33,800	33.8
3 1/2	35,900	35.9
3 3/4	38,000	38.0
4	40,100	40.1
4 1/4	42,200	42.2
4 1/2	44,300	44.3
4 3/4	46,400	46.4
5	48,500	48.5
5 1/4	50,600	50.6
5 1/2	52,700	52.7
5 3/4	54,800	54.8
6	56,900	56.9
6 1/4	59,000	59.0
6 1/2	61,100	61.1
6 3/4	63,200	63.2
7	65,300	65.3
7 1/4	67,400	67.4
7 1/2	69,500	69.5
7 3/4	71,600	71.6
8	73,700	73.7
8 1/4	75,800	75.8
8 1/2	77,900	77.9
8 3/4	80,000	80.0
9	82,100	82.1
9 1/4	84,200	84.2
9 1/2	86,300	86.3
9 3/4	88,400	88.4
10	90,500	90.5
10 1/4	92,600	92.6
10 1/2	94,700	94.7
10 3/4	96,800	96.8
11	98,900	98.9
11 1/4	101,000	101.0
11 1/2	103,100	103.1
11 3/4	105,200	105.2
12	107,300	107.3
12 1/4	109,400	109.4
12 1/2	111,500	111.5
12 3/4	113,600	113.6
13	115,700	115.7
13 1/4	117,800	117.8
13 1/2	119,900	119.9
13 3/4	122,000	122.0
14	124,100	124.1
14 1/4	126,200	126.2
14 1/2	128,300	128.3
14 3/4	130,400	130.4
15	132,500	132.5
15 1/4	134,600	134.6
15 1/2	136,700	136.7
15 3/4	138,800	138.8
16	140,900	140.9
16 1/4	143,000	143.0
16 1/2	145,100	145.1
16 3/4	147,200	147.2
17	149,300	149.3
17 1/4	151,400	151.4
17 1/2	153,500	153.5
17 3/4	155,600	155.6
18	157,700	157.7
18 1/4	159,800	159.8
18 1/2	161,900	161.9
18 3/4	164,000	164.0
19	166,100	166.1
19 1/4	168,200	168.2
19 1/2	170,300	170.3
19 3/4	172,400	172.4
20	174,500	174.5
20 1/4	176,600	176.6
20 1/2	178,700	178.7
20 3/4	180,800	180.8
21	182,900	182.9
21 1/4	185,000	185.0
21 1/2	187,100	187.1
21 3/4	189,200	189.2
22	191,300	191.3
22 1/4	193,400	193.4
22 1/2	195,500	195.5
22 3/4	197,600	197.6
23	199,700	199.7
23 1/4	201,800	201.8
23 1/2	203,900	203.9
23 3/4	206,000	206.0
24	208,100	208.1
24 1/4	210,200	210.2
24 1/2	212,300	212.3
24 3/4	214,400	214.4
25	216,500	216.5
25 1/4	218,600	218.6
25 1/2	220,700	220.7
25 3/4	222,800	222.8
26	224,900	224.9
26 1/4	227,000	227.0
26 1/2	229,100	229.1
26 3/4	231,200	231.2
27	233,300	233.3
27 1/4	235,400	235.4
27 1/2	237,500	237.5
27 3/4	239,600	239.6
28	241,700	241.7
28 1/4	243,800	243.8
28 1/2	245,900	245.9
28 3/4	248,000	248.0
29	250,100	250.1
29 1/4	252,200	252.2
29 1/2	254,300	254.3
29 3/4	256,400	256.4
30	258,500	258.5
30 1/4	260,600	260.6
30 1/2	262,700	262.7
30 3/4	264,800	264.8
31	266,900	266.9
31 1/4	269,000	269.0
31 1/2	271,100	271.1
31 3/4	273,200	273.2
32	275,300	275.3
32 1/4	277,400	277.4
32 1/2	279,500	279.5
32 3/4	281,600	281.6
33	283,700	283.7
33 1/4	285,800	285.8
33 1/2	287,900	287.9
33 3/4	290,000	290.0
34	292,100	292.1
34 1/4	294,200	294.2
34 1/2	296,300	296.3
34 3/4	298,400	298.4
35	300,500	300.5
35 1/4	302,600	302.6
35 1/2	304,700	304.7
35 3/4	306,800	306.8
36	308,900	308.9
36 1/4	311,000	311.0
36 1/2	313,100	313.1
36 3/4	315,200	315.2
37	317,300	317.3
37 1/4	319,400	319.4
37 1/2	321,500	321.5
37 3/4	323,600	323.6
38	325,700	325.7
38 1/4	327,800	327.8
38 1/2	329,900	329.9
38 3/4	332,000	332.0
39	334,100	334.1
39 1/4	336,200	336.2
39 1/2	338,300	338.3
39 3/4	340,400	340.4
40	342,500	342.5
40 1/4	344,600	344.6
40 1/2	346,700	346.7
40 3/4	348,800	348.8
41	350,900	350.9
41 1/4	353,000	353.0
41 1/2	355,100	355.1
41 3/4	357,200	357.2
42	359,300	359.3
42 1/4	361,400	361.4
42 1/2	363,500	363.5
42 3/4	365,600	365.6
43	367,700	367.7
43 1/4	369,800	369.8
43 1/2	371,900	371.9
43 3/4	374,000	374.0
44	376,100	376.1
44 1/4	378,200	378.2
44 1/2	380,300	380.3
44 3/4	382,400	382.4
45	384,500	384.5
45 1/4	386,600	386.6
45 1/2	388,700	388.7
45 3/4	390,800	390.8
46	392,900	392.9
46 1/4	395,000	395.0
46 1/2	397,100	397.1
46 3/4	399,200	399.2
47	401,300	401.3
47 1/4	403,400	403.4
47 1/2	405,500	405.5
47 3/4	407,600	407.6
48	409,700	409.7
48 1/4	411,800	411.8
48 1/2	413,900	413.9
48 3/4	416,000	416.0
49	418,100	418.1
49 1/4	420,200	420.2
49 1/2	422,300	422.3
49 3/4	424,400	424.4
50	426,500	426.5
50 1/4	428,600	428.6
50 1/2	430,700	430.7
50 3/4	432,800	432.8
51	434,900	434.9
51 1/4	437,000	437.0
51 1/2	439,100	439.1
51 3/4	441,200	441.2
52	443,300	443.3
52 1/4	445,400	445.4
52 1/2	447,500	447.5
52 3/4	449,600	449.6
53	451,700	451.7
53 1/4	453,800	453.8
53 1/2	455,900	455.9
53 3/4	458,000	458.0
54	460,100	460.1
54 1/4	462,200	462.2
54 1/2	464,300	464.3
54 3/4	466,400	466.4
55	468,500	468.5
55 1/4	470,600	470.6
55 1/2	472,700	472.7
55 3/4	474,800	474.8
56	476,900	476.9
56 1/4	479,000	479.0
56 1/2	481,100	481.1
56 3/4	483,200	483.2
57	485,300	485.3
57 1/4	487,400	487.4
57 1/2	489,500	489.5
57 3/4	491,600	491.6
58	493,700	493.7
58 1/4	495,800	495.8
58 1/2	497,900	497.9
58 3/4	500,000	500.0
59	502,100	502.1
59 1/4	504,200	504.2
59 1/2	506,300	506.3
59 3/4	508,400	508.4
60	510,500	510.5
60 1/4	512,600	512.6
60 1/2	514,700	514.7
60 3/4	516,800	516.8
61	518,900	518.9
61 1/4	521,000	521.0
61 1/2	523,100	523.1
61 3/4	525,200	525.2
62	527,300	527.3
62 1/4	529,400	529.4
62 1/2	531,500	531.5
62 3/4	533,600	533.6
63	535,700	535.7
63 1/4	537,800	537.8
63 1/2	539,900	539.9
63 3/4	542,000	542.0
64	544,100	544.1
64 1/4	546,200	546.2
64 1/2	548,300	548.3
64 3/4	550,400	550.4
65	552,500	552.5
65 1/4	554,600	554.6
65 1/2	556,700	556.7
65 3/4	558,800	558.8
66	560,900	560.9
66 1/4	563,000	563.0
66 1/2	565,100	565.1
66 3/4	567,200	567.2
67	569,300	569.3
67 1/4	571,400	571.4
67 1/2	573,500	573.5
67 3/4	575,600	575.6
68	577,700	577.7
68 1/4	579,800	579.8
68 1/2	581,900	581.9
68 3/4	584,000	584.0
69	586,100	586.1
69 1/4	588,200	588.2
69 1/2	590,300	590.3
69 3/4	592,400	592.4
70	594,500	594.5
70 1/4	596,600	596.6
70 1/2	598,700	598.7
70 3/4	600,800	600.8
71	602,900	602.9
71 1/4	605,000	605.0
71 1/2	607,100	607.1
71 3/4	609,200	609.2
72	611,300	611.3
72 1/4	613,400	613.4
72 1/2	615,500	615.5
72 3/4	617,600	617.6
73	619,700	619.7
73 1/4	621,800	621.8
73 1/2	623,900	623.9
73 3/4	626,000	626.0
74	628,100	628.1
74 1/4	630,200	630.2
74 1/2	632,300	632.3
74 3/4	634,400	634.4
75	636,500	636.5
75 1/4	638,600	638.6
75 1/2	640,700	640.7
75 3/4	642,800	642.8
76	644,900	644.9
76 1/4	647,000	647.0
76 1/2	649,100	649.1
76 3/4	651,200	651.2
77	653,300	653.3
77 1/4	655,400	655.4
77 1/2	657,500	657.5
77 3/4	659,600	659.6
78	661,700	661.7
78 1/4	663,800	663.8
78 1/2	665,900	665.9
78 3/4	668,000	668.0
79	670,100	670.1
79 1/4	672,200	672.2
79 1/2	674,300	674.3
79 3/4	676,400	676.4
80	678,500	678.5
80 1/4	680,600	680.6
80 1/2	682,700	682.7
80 3/4	684,800	684.8
81	686,900	686.9
81 1/4	689,000	689.0
81 1/2	691,100	691.1
81 3/4	693,200	693.2
82	695,300	695.3
82 1/4	697,400	697.4
82 1/2	699,500	699.5
82 3/4	701,600	701.6
83	703,700	703.7
83 1/4	705,800	705.8
83 1/2	707,900	707.9
83 3/4	710,000	710.0
84	712,100	712.1
84 1/4	714,200	714.2
84 1/2	716,300	716.3



Case No.	Site	A	B	C	D	E	F	G	H	I	J	K	L	M	N	O	P	Q	R	S	T	U	V	W	X	Y	Z	2010 Normal	On the edge of the normal
Normal	27.4	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2	1.2
1	100	8	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
2	327	210	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
3	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
4	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
5	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
6	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
7	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
8	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
9	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
10	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
11	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
12	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
13	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
14	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
15	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
16	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
17	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
18	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
19	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
20	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
21	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
22	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
23	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
24	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
25	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
26	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
27	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
28	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
29	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
30	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
31	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
32	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
33	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
34	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
35	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
36	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
37	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
38	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
39	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
40	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
41	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
42	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
43	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
44	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
45	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
46	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
47	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
48	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
49	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
50	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
51	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
52	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
53	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
54	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
55	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
56	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
57	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
58	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
59	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
60	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
61	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
62	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
63	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
64	100	10	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1									

Manuscript—Wang, Fong and Li-Tsun Hsu  
 File 1 in Columbia University.



SEASON	I	II	III	IV	V	VI	VII	VIII	IX	X	Weight lbs.	Antagonist (Real Periods)	U. S. Title
National Nov. 1920	2-21	1-20	20	22	11	19	20	12	0-0	1-20	1,200	Future in doubt	
National Nov. 1921	2-20	0-23	40	33	19	15	31	30	0-0	1-20	1,000	Yield to heart	
National Feb. 1922	2-08	2-12	45	37	22	22	33	35	20	0-0	900	Yield to heart	
National Jan. 1923	2-06	2-03	34	36	22	22	33	30	21	2-20	1,000	Loss of consciousness	
National Nov. 1923	2-03	2-03	24	31	21	19	27	30	30	0-0	400	Yield to heart	
A. J. Meyer Nov. 5	2-20	1-20	30	21	11	19	19	12	0-0	1-20	1,000	Yield to heart Adversely injured	
A. J. Meyer Jan. 4	0-00	1-21	52	40	31	33	28	0-0	0-00	1-20	1,000	Yield to heart	
A. J. Meyer Nov. 8	1-00	1-21	32	23	19	20	31	33	14	1-20	1,000	Yield to heart	
A. J. Meyer	0-00					30			11-1	0-00	0-00	Yield to heart	
A. J. Meyer	4-00	2-20	62	50	31	33	27	33	30	0-0	1,000	Yield to heart	

Estimated load based upon average strength of yield point of barrel of 200 lb. per sq. in., average tensile strength barrel 21,000 lb. per sq. in., average strength of strand 140,000 lb. per sq. in.

FIG. 3.—Males. Scaleless. Headless and Body Translucent.

### Large Leaves

Yours	Concurrence	Sharing
Personal (your point)	Ultimate (with point)	Ultimate (with point)
20,000 30,000	70,000 40,000	100,000 30,000
Multiple of majority (divided)		20,000 30,000
Multiple of majority (following)		1,000,000
Multiple of majority (with and without share)		
Share (without and following share) 20,000 30,000		

## Strength of Special Steel Alloys—Lbs. per Square Inch

[illegible]

Strength of Copper, Aluminum and Various Alloys—  
Lbs. per Square Inch

Case 1:  $\alpha = 0$

Students	27,000
Faculty/staff	5,000
Operating	10,000
Student of residence (dorms)	1,000,000
Student of campus, library	5,000,000
Student of campus (university) or (state)	20,000

## Cold Rolled or Annealed Plate

Expenditure	27 000
Capital grant	100 000
Net change	73 000

## Field strength 33.34

Number of iterations of each of 4 groups of 1000 runs: 1 group of 1000 runs and 3 groups of 1000 runs.

*Journal of Management Education* 32(1)

Length	14.000
Length at 50%	12.000
Standard error	1.000

## 4-ethyl-3-methyl-5-norbornene-2-ene (1) and 5-ethyl-

<p> <math>L = 1</math> mm  <math>\Delta t = 1</math> ms          Spatial step of observation, <math>\Delta x = 1</math> mm       </p>	<p>         Frequency, <math>f = 1</math> Hz  <math>\Delta t = 1/20</math> ms       </p>
---	--

*Affiliate marketing*

[illegible]

The steel industry's present reluctance to change manufacturing practices is the solid steel sheet generally designated as cold rolled steel (CLRS). While the use of warm working transfers us to the third of the three stages, it does not take us beyond the second stage to order. It is a pity to find that the contingencies and uncertainties of Government are undermining its drive with the steel industry to change. The industry has been brought to our attention when upon the release of some small stamped flings, the apparently small change in the thickness of the flings, the layers of thinner material together add up to a few pounds of steel. The industry is to pump too quickly to the steel extreme and attempt to meet the customer's strength alloy—requiring extra working time and further heat treatment. The industry is to be commended in its willingness to produce to customer's needs, but the added responsibility and cost of extra design work upon inexperienced buyers would be

The following table outlines the general influence of chemical composition of the physical properties of steel:

[illegible][illegible]

The above table, while very comprehensive, should not be considered as final.

The matter of weldability of the chrome-nickel alloys is not dealt with in this paper. However, very valuable information shows that 3% nickel model steels give better welds than the chrome-nickel steels.

The S.A.E. specification No. 3030 for a hot carbon-chrome-iron steel, or Specification No. 2330 for 3½ nickel steel would seem to meet the requirements of the manufacturers as well as the Army specifications of the S.A.E. No. 413 (chrome-nickel), assuming at the same time, the great danger of segregation due to badly heat-treated—

First treatment and its influence cannot be given fully, requires very careful study and intense care in application.

## Strength and Weight of Mild Steel Rivets and Pins

Distance (miles)	Strength in Pounds	Crushing = F
0	1,000	1,000
10	800	750
20	600	500
30	400	250
40	200	100
50	100	50
60	50	25
70	25	10
80	10	5
90	5	2
100	0	0

The above values are based upon a bearing strength of parallel joint square inch  $f_v = 22,000$  and a crushing strength  $f_c = 26,000$ .

$$D_1 = f_1 \frac{r_1^2}{4} \text{ in.}^2 \text{ say } 0.001 \text{ in.}^2 \text{ when}$$

The values for crushing have been worked out for a plate 1 mm thick, then here the crushing strength for various thicknesses of plate may be computed by multiplying the above values by  $\frac{1}{2}t^2$  or  $\frac{1}{2}t^3$ .

If the crushing strength of the rivet is greater than its shearing strength the design should be based upon the smaller result.

In case the soil is subjected to a load other than tension the strength should be based upon the corresponding form of loading. When bolts and pins are used in turnbuckle fittings and wire connections they are usually subjected to a form of bending and should be calculated as a beam round cross section loaded at the center.

$$f = \frac{M}{l}, \text{ where } l = \text{modulus of rigidity, } M = \text{bending mo}$$

such due to load generally considered concentrated at the center,  $l$  = distance from neutral axis to most stressed fiber in this case  $1/2 D$ ,  $I$  = moment of inertia of cross section.

As differing bits that involve wooded materials in failure of the wood should be considered first, since in this type of nonuniform rupture is most often caused by the fastening pulling out or loosening due to the wood cracking in front of the bolts. The crushing load may be computed from the formula  $P = L \cdot N \cdot C \cdot D$ . Where  $P$  = crushing load,  $L$  = crushing





### Darsenborg Reinsured

The Darsenborg Motor Corp. has secured the Darsenborg Motor Co., St. Paul, as the new Vice President of Chicago, its capital being \$1,000,000 of which \$500,000 is paid in. The new group has accepted a place in the East in

### Yale Lost Officially Reinsured

The Yale Motor has been reinsured as Arthur Reed, President No. 1, a voluntary reinsurance corporation, has entered in the Second National Bank of New York City. The Yale Motor has been reinsured as Arthur Reed, President No. 1, a voluntary reinsurance corporation, has entered in the Second National Bank of New York City.

### Heart Jolted with Goodrich

Heart Jolted with Goodrich is the name of the company operating the motor of the airplane which the movement is being built by the R. J. Goodrich Co.

The United States will need many airplanes to maintain its position in the world. The larger airplane is not in use for 100 miles and will be in use for 100 miles.

Because the airplane will be used in the future, it is necessary to have a large number of airplanes in the future.

The airplane will be used in the future, it is necessary to have a large number of airplanes in the future.

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The airplane will be used in the future, it is necessary to have a large number of airplanes in the future.



The above photograph shows the latest pattern of the Curtiss Model A, and the engine. The engine is a four-cylinder, 100-horsepower, and the propeller is a three-bladed, 10-foot diameter.

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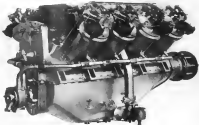
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BUILT BY THE CURTISS AEROPLANE & MOTOR CORPORATION

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1000 1200 1400 1600 1800 2000 2200 2400 2600 2800 3000

INDICATED HORSEPOWER PER MINUTE



*Curtiss*

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